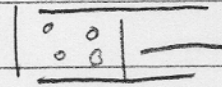


## Last Time

① Work

work done by gas

$$dW = p dV$$



gas expands  $dW > 0$

gas compressed  $dW < 0$

work done on gas  $> 0$

② The first law:

$$\Delta U = Q - W$$

$$dU = dQ - p dV$$

③ For ideal gasses used the first law

To show:

$$C_p = C_v + R$$

$$dQ_p = n C_p \Delta T \quad \leftarrow \begin{array}{l} \text{at} \\ \text{const press., temp changes} \\ \text{when heat added} \end{array}$$

$$dQ_v = n C_v \Delta T \quad \leftarrow \begin{array}{l} \text{const vol, temp changes} \\ \text{when heat added} \end{array}$$

Used this to show for a mono-atomic ideal gas

MAIG :

$$C_V = \frac{3}{2} R$$

$$C_P = \frac{5}{2} R$$

definition = "the adiabatic index"

$$\gamma = \frac{C_P}{C_V} = \frac{5}{3}$$

For diatomic IC

DAIG

$$C_V = \frac{5}{2} R$$

$$C_P = \frac{7}{2} R$$

$$\gamma = \frac{C_P}{C_V} = \frac{7}{5}$$

notice  $\gamma > 1$

"  $\gamma$  always

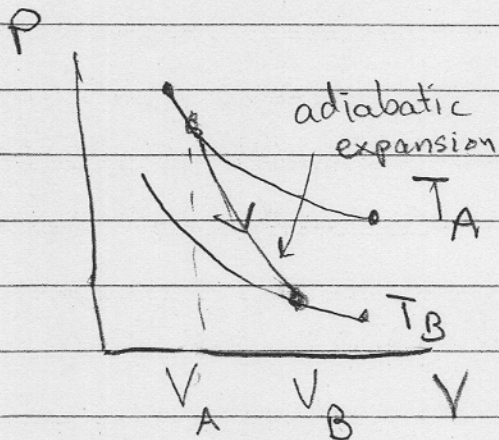
④ Types of expansions  $\leftrightarrow$  computed

- a) isothermal  $T = \text{Const}$
- b) isobaric  $P = \text{Const}$
- c) const vol.  $V = \text{Const}$
- d) Adiabatic  $Q = 0$

Adiabatic Expansion  $Q=0$  ← no heat influx

$$dU = dQ - dW$$

$dU = -dW$  ← the internal energy of the gas is used to do work



① In general if a system expands heat would be needed to be added to maintain the temperature.

② In an Adiabatic expansion,  $Q=0$  then the temperature decreases

③ An analysis of the first law, shows (see following pages)

$$PV^\gamma = C \quad C = \text{constant}$$

Calculate the Work Done going from  $V_A \rightarrow V_B$ :

$$W = \int P dV$$

$$PV^\gamma = P_A V_A^\gamma \Rightarrow P = P_A \left(\frac{V_A}{V}\right)^\gamma$$

$$W = \int P_A \left(\frac{V_A}{V}\right)^\gamma dV$$

So

$$W = \int_{V_A}^{V_B} P_A V_A \left(\frac{V_A}{V}\right)^\gamma \frac{dV}{V_A} \quad \text{let } u \equiv \frac{V}{V_A}$$

$$W = P_A V_A \int_{u=1}^{u=V_B/V_A} du u^{-\gamma}$$

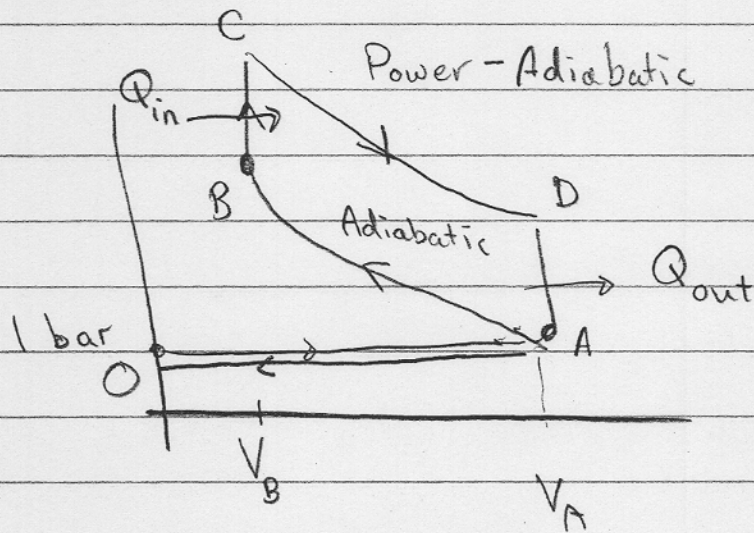
$$= P_A V_A \left. \frac{u^{-\gamma+1}}{-\gamma+1} \right|_1^{V_B/V_A}$$

$$W = \frac{P_A V_B}{-\gamma+1} \left[ \left(\frac{V_B}{V_A}\right)^{-\gamma+1} - 1 \right]$$

notice  $-\gamma+1 < 0$  a "hidden minus"

$$W = \frac{P_A V_A}{(\gamma-1)} \left[ 1 - \left(\frac{V_A}{V_B}\right)^{\gamma-1} \right]$$

# Otto Cycle:



$$\frac{V_A}{V_B} = \text{"The compression"} = \gamma = 8$$

factor

Treat as a diatomic gas  $\gamma = 1.4$

## The Otto Cycle - We will analyze @ 4 Problems

• A four stroke engine - see video

① Pull in air-gas mixture (OA)  
- gas brought in at  $T \approx 300^\circ\text{K}$  and  $P = 1\text{atm}$

② Compress the mixture (AB) - Adiabatic  
- for argument take  $\frac{V_A}{V_B} = \text{the compression ratio} = 8$

③ Ignite the mixture (BC)  
- volume remains constant  $Q_{in}$  comes from burning gas

④ Power Stroke - Adiabatic (CD)

⑤ Exhaust Valve - opened (DA)  
- Pressure down to atmospheric

⑥ Cylinder compressed AO

happens  $\sim 1000$  times  $t \sim 0.06\text{s}$   
min

Probl - estimate the volume of gas sucked in, estimate the number of moles

$$V \sim 3.0 \text{ L}$$

$$pV = nRT \Rightarrow n = \frac{pV}{RT} = 0.13$$

$$P = 1 \text{ bar} = 10^5 \text{ N/m}^2$$

$$V = 3 \times 10^{-3} \text{ m}^3$$

$$R = 8.3 \quad T = 300^\circ \text{K}$$

Probl - estimate the rise in temp of the compression stroke

Sol: Since its adiabatic

$$PV^\gamma = C \quad \text{now} \quad PV = nRT$$

$$P = \frac{nRT}{V}$$

$$\frac{nRT}{V} V^\gamma = C$$

a useful result

So

$$\boxed{TV^{\gamma-1} = C'} \quad \leftarrow \text{another const}$$

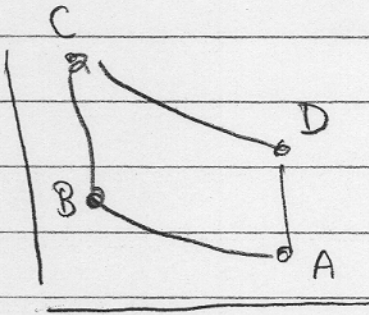
$$T_B V_B^{\gamma-1} = T_A V_A^{\gamma-1}$$

$$\frac{T_B}{T_A} \approx \left( \frac{V_A}{V_B} \right)^{\gamma-1} = r^{\gamma-1} = (8)^{0.4} \approx 2.29$$

So

$$T_B = 300^\circ \text{K} \cdot 2.29 = 689^\circ \text{K} \sim 750^\circ \text{F}$$

Notice: CD similar to BA



$$\frac{T_B}{T_A} = \left( \frac{V_A}{V_B} \right)^{\gamma-1} = r^{\gamma-1}$$

$$\frac{T_C}{T_D} = \left( \frac{V_C}{V_D} \right)^{\gamma-1} = r^{\gamma-1}$$

1  
The same as  $(T_B/T_A)$



## Engine Construction

① Four Cylinders tied to a crank shaft.

② How do the valves know how to open and close?

- The cam is a long metal shaft - which by rotating opens and closes the valves

### Problem 3

The efficiency is defined as the work done per unit heat input

$$\epsilon = \frac{W}{Q_{in}} = \frac{Q_{in} - Q_{out}}{Q_{in}} = 1 - \frac{Q_{out}}{Q_{in}}$$

Since:  $\Delta U = Q - W$        $Q = Q_{in} - Q_{out}$

$$Q_{in} - Q_{out} = W$$

Want to determine the efficiency,

① Show that:

$$\epsilon = 1 - \left( \frac{V_B}{V_A} \right)^{\gamma-1}$$

$$\frac{V_A}{V_B} \equiv \text{the compression ratio} \equiv r \sim 8$$

$$\epsilon = 1 - \frac{1}{r^{\gamma-1}}$$

Solution:

Analyze the heat brought in

$$\Delta U = Q - W$$

$$n C_V dT = dQ$$

$$Q_H = nC_V (T_C - T_B)$$

Now  $Q_L$

$$\Delta U = Q = \cancel{W}$$

heat flowing into gas

$$\Delta U = nC_V (T_A - T_D) = Q$$

$$T_A < T_D$$

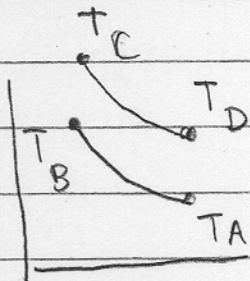
So

$$Q_{\text{out}} = -Q = nC_V (T_D - T_A)$$

Then

$$\epsilon = \frac{W}{Q_{\text{out}}} = 1 - \frac{Q_{\text{out}}}{Q_{\text{in}}} = 1 - \left[ \frac{T_D - T_A}{T_C - T_B} \right] \frac{nC_V}{nC_V}$$

Now we need to relate



$T_C$  and  $T_D$  and  $T_B$  and  $T_A$

Using

$$T_c = T_D r^{\gamma-1} \quad T_B = T_A r^{\gamma-1}$$

So

$$e = 1 - \left[ \frac{T_D - T_A}{T_D r^{\gamma-1} - T_A r^{\gamma-1}} \right]$$

$$e = 1 - \frac{1}{r^{\gamma-1}} = 1 - \frac{1}{8^{0.4}} = 0.56$$

S