

Last Time

① Finished up quasi-statics, i.e. where c is fast compared to $\frac{L}{T}$, or

where $\omega \ll c/L$, by studying

quasi-statics in metals

② Metals have their own frequency scales:

• $\frac{1}{\tau_c}$ the frequency of collisions with impurities and each other $\frac{1}{\tau_c} \sim 10^{14} \text{ Hz}$

• $\omega_p^2 \approx \frac{ne^2}{m}$ The plasma frequency.
Natural coulomb oscillation / orbital frequency of electrons

$$\sim \frac{13.6 \text{ eV}}{\hbar} \sim 10^{16} \text{ Hz}$$

• The conductivity is in a classical ^{Drude} model of electron transport:

$$\sigma = \omega_p^2 \tau_c = \frac{ne^2 \tau_c}{m}$$

$$\sim \omega_p \underbrace{(\omega_p \tau_c)}_{\sim 100} \sim 10^{18} \frac{1}{\text{s}}$$

③ For quasi-statics in metals we need

$$\omega \ll c/L \leftarrow \text{quasi-static}$$

$$\left. \begin{array}{l} \omega \ll 1/\tau_c \\ \omega \ll \omega_p \end{array} \right\} \leftarrow \omega \text{ less than micro frequencies}$$

Saw that a new scale arose:

ω			quasi-static
$\omega < \omega_{ind}$	induced currents are small	induced currents screen magnetic fields on scale	macroscopic
	magnetic fields are not screened on length L		$E+M$ doesn't apply
		$\delta = \sqrt{\frac{2c}{\omega \sigma}}$	$\omega \sim 1/\tau_c$
		$\delta < L$	$\omega \sim \frac{L}{c}$
			$\omega \sim \omega_p$

$$\omega_{ind} \sim \frac{c^2}{L^2 \delta}$$

④ Formally showed that B satisfies a diffusion equation: because frequencies are large,

$$-\nabla^2 B = \frac{\sigma}{c^2} \partial_t B$$

These terms are equal when $\omega \sim \omega_{\text{ind}}$

$$\frac{B}{L^2} \sim \frac{\sigma \omega B}{c^2}$$

$$\boxed{\frac{c^2}{L^2 \sigma} \sim \omega}$$

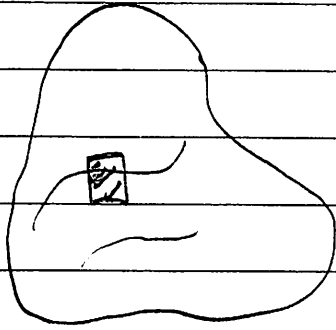
ω_{ind}

Today

- Summarize Maxwell equations and talk about waves

Energy Conservation for Simple Matter

The energy density



• $u_{\text{TOT}} \equiv$ the total energy per volume

• $u_{\text{EM}} \equiv$ the electric + magnetic energy per vol

$$u_{\text{EM}} = \underbrace{\frac{1}{2} \vec{E} \cdot \vec{D}}_{= u_{\text{EM}}^{\text{E}}} + \underbrace{\frac{1}{2} \vec{H} \cdot \vec{B}}_{= u_{\text{EM}}^{\text{B}}}$$

• $u_{\text{mech}} =$ mechanical energy per vol.

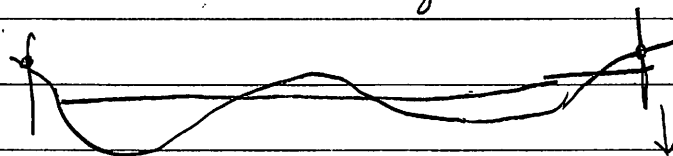
• The ^{Total} energy flux, \vec{S}_{TOT} . Expect:

$$\partial_t u_{\text{TOT}} + \partial_i S_{\text{TOT}}^i = 0$$

In this way an isolated system will conserve energy

• The Energy flux has mechanical pieces: S_{mech}

→ Take a stretched string for example



Force x velocity

$$S_{\text{string}} = T_0 \frac{\partial y}{\partial x} \frac{\partial y}{\partial t}$$

→ S_{mech} records how energy is ^{mechanically} transported from one part to the other;

• The electromagnetic flux S_{em} :

$$S_{\text{em}} = c (\vec{E} \times \vec{H})$$

↑ Poynting vector

So expect:

$$\partial_t (U_{\text{mech}} + U_{\text{em}}) + \partial_i (S_{\text{mech}}^i + S_{\text{em}}^i) = 0$$

In integral form: $U = \int u \, dV$

$$\frac{d(U_{\text{mech}} + U_{\text{em}})}{dt} = - \int \vec{S}_{\text{em}} \cdot d\vec{a} - \int \vec{S}_{\text{mech}} \cdot d\vec{a}$$

0 for an isolated mechanical system

Prf

$$\partial_t u_{\text{mech}} + \partial_i S^i_{\text{mech}} = \underbrace{\vec{j} \cdot \vec{E}}_{\text{work done per vol}}$$

Now

$$\begin{aligned} \vec{j} \cdot \vec{E} &= c (\nabla \times \vec{H} - \frac{1}{c} \partial_t \vec{D}) \cdot \vec{E} \\ &= \underbrace{c (\nabla \times \vec{H} \cdot \vec{E})}_{c \nabla \cdot (\vec{H} \times \vec{E}) + H c \nabla \times \vec{E}} - \vec{E} \cdot \partial_t \vec{D} \end{aligned}$$

$$= -\nabla \cdot \vec{S}_{\text{em}} - H (\partial_t B) - E \partial_t D$$

$$\vec{j} \cdot \vec{E} = -\nabla \cdot \vec{S}_{\text{em}} - \underbrace{(E \partial_t D + H \partial_t B)}_{\partial_t u_{\text{em}}}$$

So

$$\vec{j} \cdot \vec{E} = -\nabla \cdot \vec{S}_{\text{em}} - \partial_t u_{\text{em}}$$

And

$$\partial_t u_{\text{mech}} + \partial_i S^i = -\partial_t u_{\text{em}} - \partial_i S^i_{\text{em}}$$

Momentum Conservation :

$$\partial_t g_{\text{TOT}}^j + \partial_i T_{\text{TOT}}^{ij} = 0$$

a conservation law, then

If \vec{g} obeys a total momentum of an isolated system is conserved:

g_{TOT}^j = is the momentum per volume

T^{ij} = is the force in the j -th direction per area in the i -th direction.

$$\frac{dP_{\text{TOT}}^j}{dt} = \int dV \partial_t g_{\text{TOT}}^j = \int -\partial_i T^{ij} dV$$

$$= \int_{\partial V} T^{ij} \vec{n}_i dS = 0$$

for an "isolated" system

$$\bullet g_{\text{TOT}}^j = g_{\text{mech}}^j + g_{\text{em}}^j$$

$$\bullet T_{\text{TOT}}^{ij} = T_{\text{mech}}^{ij} + T_{\text{em}}^{ij}$$

$$\vec{g}_{\text{em}} = \frac{\epsilon_0 \mu_0}{c^2} \vec{S}_{\text{em}}$$

$$\bullet T_{\text{em}}^{ij} = \frac{-1}{2} (E^i D^j + D^i E^j - \vec{D} \cdot \vec{E} \delta^{ij}) + \frac{-1}{2} (H^i B^j + H^j B^i - \vec{H} \cdot \vec{B} \delta^{ij})$$

$$= \epsilon_0 \left(-E^i E^j + \frac{1}{2} E^2 \delta^{ij} \right) + \frac{1}{\mu_0} \left(-B^i B^j + \frac{1}{2} B^2 \delta^{ij} \right)$$

So the full result is

$$\partial_t g_{\text{mech}} + \partial_t g_{\text{em}} + \partial_i T_{\text{mech}}^{i\alpha} + \partial_i T_{\text{em}}^{i\alpha} = 0$$

For an mechanically isolated system, $\vec{P} = \int \vec{g} dV$

$$\frac{d}{dt} (P_{\text{mech}} + P_{\text{em}}) = - \int dS n_i T_{\text{em}}^{i\alpha} - \int dS n_i T_{\text{mech}}^{i\alpha}$$

zero
for
mechanically

isolated systems

Recall that $-\partial_i T_{mech}^{ij}$ is the force per volume $\equiv \vec{f}_{mech}$
 So this says

$$\frac{\partial g_{mech}^j}{\partial t} + \underbrace{\frac{\partial T_{mech}^{ij}}{\partial x^i}}_{\vec{f}_{mech}^j} = \vec{f}_{em}^j$$

Prf

$$\vec{f}_{em} = \rho \vec{E} + \left(\vec{j} \times \vec{B} \right)^j$$

Now write, $\rho = \nabla \cdot \vec{D}$ and $\vec{j} = \nabla \times \vec{H} - \frac{1}{c} \partial_t \vec{D}$,

Find

$$\vec{f}_{em}^j = \underbrace{(\nabla \cdot \vec{D}) E^j}_{(1)} + \underbrace{[(\nabla \times \vec{H}) \times \vec{B}]^j}_{(2)} - \underbrace{\frac{1}{c} (\partial_t \vec{D} \times \vec{B})^j}_{(3)}$$

The rest is labor and I will not go through

$$(1) \Rightarrow -\partial_i T_E^{ij} \quad \text{with}$$

$$T_E^{ij} = -\epsilon E^i E^j + \frac{1}{2} E^2 \delta^{ij}$$

$$(2) \Rightarrow -\partial_i T_B^{ij} \quad \text{with}$$

$$T_B^{ij} = -\frac{1}{\mu} B^i B^j + \frac{1}{2\mu} B^2 \delta^{ij}$$

$$(3) \Rightarrow -\partial_t \vec{g}$$

$$\vec{g}_{em} = \frac{\epsilon \mu}{c} \vec{E} \times \vec{H}$$

Angular Momentum Conservation

$$\partial_t g^k + \partial_l T^{lk} = 0 \quad \Leftarrow \text{momentum conservation}$$

And take $T^{lk} = T^{kl}$ symmetric

$$\partial_t ((\vec{r} \times \vec{g})_i) + \epsilon_{ijk} r^j \partial_l T^{lk} = 0$$

ang
momentum

$$\frac{\partial r^j}{\partial r^l}$$

Now $\epsilon_{ijk} r^j \partial_l T^{lk} = \epsilon_{ijk} \left[\partial_l (r^j T^{lk}) - \underbrace{\delta_l^j}_{=T^{jk}} T^{lk} \right]$

So since $\epsilon_{ijk} T^{jk} = 0$, we have

$$\partial_t ((\vec{r} \times \vec{g})_i) + \partial_l (\epsilon_{ijk} r^j T^{lk}) = 0$$

So the angular momentum of the system is conserved, provided the stress tensor is symmetric

$$\vec{L}_{\text{field}} = \int_V \vec{r} \times \vec{g}_{em}$$

$$\frac{d}{dt} (\vec{L}_{\text{mech}} + \vec{L}_{\text{field}})_i = - \underbrace{\int_{\partial V} dS \epsilon_{ijk} r^j T^{kl} n_l}_{\text{net torque exerted on system}}$$

net torque
exerted on system